## KW Calculations

## Calculating KW Requirements for Heating Liquids and Gases

When calculating the required KW , always use the maximum flow of the medium to be heated, the minimum temperature at the heater inlet, and the maximum desired outlet temperature. Also include a $20 \%$ Safety Factor to allow for heat losses to jacket and piping, voltage variation and wattage tolerance.

For specific heat and density values see Properties of Materials Tables on page 16-4.
Safe element watt density and sheath material charts are located on pages 16-12 through 16-20.

## Formula for Heating Liquids

$\mathrm{KW}=\frac{\text { Flow } \times 60 \text { minute } / \text { hour } \times \text { Density } \times \text { Specific heat } \times \Delta \mathrm{T} \times \text { Safety factor }}{3412 \mathrm{BTU} / \mathrm{KWH}}$

Flow $=$ Flow in gallons/minute
Density = Density of liquid in pounds/gallon
Specific Heat $=$ Specific heat of liquid in BTU/pound ${ }^{\circ} \mathrm{F}$
$\Delta \mathbf{T}=$ Temperature rise in ${ }^{\circ} \mathrm{F}$

## Sample problem for heating water:

Calculate KW required to heat 5 gallons/minute of water from 50 to $100^{\circ} \mathrm{F}$.
$\mathrm{KW}=\frac{5 \mathrm{gal} / \mathrm{min} \times 60 \mathrm{~min} / \mathrm{hr} \times 8.34 \mathrm{lb} / \mathrm{gal} \times 1.0 \mathrm{BTU} / \mathrm{lb}^{\circ} \mathrm{F} \times 50^{\circ} \mathrm{F} \times 1.2}{3412 \mathrm{BTU} / \mathrm{KWH}}$
Total KW required $=44$
Water Flow Chart for Tempco 3" and 5" Flanged Circulation Heaters
Maximum water flow per hour through selected heaters at specified temperature rise.

| Part Number | $\mathbf{K W}$ | $\mathbf{2 0}^{\circ} \mathbf{F}$ | $\mathbf{3 0}^{\circ} \mathbf{F}$ | $\mathbf{4 0}^{\circ} \mathbf{F}$ | $\mathbf{5 0}{ }^{\circ} \mathbf{F}$ | $\mathbf{6 0}{ }^{\circ} \mathbf{F}$ | $\mathbf{7 0}^{\circ} \mathbf{F}$ | $\mathbf{8 0}^{\circ} \mathbf{F}$ | $\mathbf{9 0}^{\circ} \mathbf{F}$ | $\mathbf{1 0 0}^{\circ} \mathbf{F}$ | $\mathbf{1 1 0}^{\circ} \mathbf{F}$ | $\mathbf{1 2 0}^{\circ} \mathbf{F}$ | $\mathbf{1 3 0}^{\circ} \mathbf{F}$ |
| :---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| CHF01891 | 6 | 123 | 82 | 61 | 49 | 41 | 35 | 31 | 27 | 25 | 22 | 20 | 19 |
| CHF01895 | 9 | 184 | 123 | 92 | 74 | 61 | 53 | 46 | 41 | 37 | 33 | 31 | 28 |
| CHF01898 | 12 | 245 | 164 | 123 | 98 | 82 | 70 | 61 | 55 | 49 | 45 | 41 | 38 |
| CHF01901 | 15 | 307 | 205 | 153 | 123 | 102 | 88 | 77 | 68 | 61 | 56 | 51 | 47 |
| CHF01904 | 18 | 368 | 245 | 184 | 147 | 123 | 105 | 92 | 82 | 74 | 67 | 61 | 57 |
| CHF01928 | 24 | 491 | 327 | 245 | 196 | 164 | 140 | 123 | 109 | 98 | 89 | 82 | 76 |
| CHF01931 | 30 | 614 | 409 | 307 | 245 | 205 | 175 | 153 | 136 | 123 | 112 | 102 | 94 |
| CHF01934 | 36 | 736 | 491 | 368 | 295 | 245 | 210 | 184 | 164 | 147 | 134 | 123 | 113 |
| CHF01935 | 50 | 1023 | 682 | 511 | 409 | 341 | 292 | 256 | 227 | 205 | 186 | 170 | 157 |
| CHF01936 | 60 | 1227 | 818 | 614 | 491 | 409 | 351 | 307 | 273 | 245 | 223 | 205 | 189 |

(Gallons) $\mathrm{HR}=\frac{(\mathrm{KW})(3412)}{(8.34)(\Delta \mathrm{T})}$
NOTE: Safety factor not included. Add to suit application.

Formula for Heating Gases
$\mathrm{KW}=\frac{\text { Flow } \times 60 \text { minute } / \text { hour } \times \text { Density } \times \text { Specific heat } \times \Delta \mathrm{T} \times \text { Safety factor }}{3412 \mathrm{BTU} / \mathrm{KWH}}$

Flow = Flow in SCFM (standard cubic feet per minute measured at 14.7 PSIA and $70^{\circ} \mathrm{F}$ )

Density = Density of gas in pounds/cubic foot at standard conditions.

Specific Heat $=$ Specific heat of gas in BTU/pound ${ }^{\circ} \mathrm{F}$ at standard conditions.
$\Delta \mathbf{T}=$ Temperature rise in ${ }^{\circ} \mathrm{F}$

NOTE: If air flow is given in CFM at operating temperature and pressure it can be converted to SCFM (Standard Cubic Feet per Minute) with the following formula:
$\mathrm{SCFM}=\mathrm{CFM} \times \frac{\mathrm{PSIG}+14.7}{\mathrm{~T}+460} \times 35.37$
PSIG $=$ operating pressure (gauge pressure in lbs/sq.in.)
$\mathrm{T}=$ operating temperature in ${ }^{\circ} \mathrm{F}$
$\mathrm{SCFM}=$ flow rate in CFM at standard conditions of $60^{\circ} \mathrm{F}$ and 14.7 PSIA.
View Product Inventory @ www.tempco.com

## Calculating KW Requirements to Superheat Steam



## Superheated Steam Graph

## Problem: Heat $420 \mathrm{lbs} / \mathrm{hr}$ of $90 \%$ quality steam to $620^{\circ}$ @ 75PSIG

1. Plot the pressure on graph $\mathbf{P}$ and the steam quality on graph Q . Draw a straight line from $\mathbf{P}$ through $\mathbf{Q}$ and read W1.
2. Plot the degrees of superheat on graph S . The degrees of superheat equals operating temperature minus saturated temperature. Saturated temperature is read beside gauge pressure on graph P . $620^{\circ} \mathrm{F}-320^{\circ} \mathrm{F}=300^{\circ} \mathrm{F}$
Draw a straight line from P through S and read W 2 .
3. Determine the required KW using the following equation:
$\mathrm{KW}=\mathrm{LBS} / \mathrm{HR} \times(\mathrm{W} 2-\mathrm{W} 1) / 1000 \times$ Safety factor $=420 \times(114-36) / 1000 \times 1.2=39.3 \mathrm{KW}$

Note: Element watt density is critical in choosing the correct circulation heater and is dependent upon maximum operating temperature and steam velocity.

Standard Pipe Data

| Nominal <br> Pipe <br> Size | Threads <br> Per <br> Inch | Inside <br> Diameter <br> (inches) | Outside <br> Diameter <br> (inches) | Weight <br> Pipe <br> (lbs/ft) | Length in Feet <br> Containing One <br> Cubic Foot | Gallons in <br> One Linear <br> Foot | Weight <br> Water <br> (Ibs/ft of Pipe) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $1 / 8$ | 27 | 0.269 | 0.405 | 0.244 | 2526.000 | 0.0030 | 0.025 |
| $1 / 4$ | 18 | 0.364 | 0.540 | 0.424 | 1383.800 | 0.0054 | 0.045 |
| $3 / 8$ | 18 | 0.493 | 0.675 | 0.567 | 754.360 | 0.0099 | 0.083 |
| $1 / 2$ | 14 | 0.622 | 0.840 | 0.850 | 473.910 | 0.0158 | 0.132 |
| $3 / 4$ | 14 | 0.824 | 1.050 | 1.130 | 270.030 | 0.0277 | 0.231 |
| 1 | $111 / 2$ | 1.049 | 1.315 | 1.678 | 166.620 | 0.0449 | 0.374 |
| $11 / 4$ | $111 / 2$ | 1.380 | 1.660 | 2.272 | 96.275 | 0.0777 | 0.648 |
| $11 / 2$ | $111 / 2$ | 1.610 | 1.900 | 2.717 | 70.733 | 0.1058 | 0.882 |
| 2 | $111 / 2$ | 2.067 | 2.375 | 3.652 | 49.913 | 0.1743 | 1.453 |
| $21 / 2$ | 8 | 2.469 | 2.875 | 5.793 | 30.077 | 0.2487 | 2.073 |
| 3 | 8 | 3.068 | 3.500 | 7.575 | 19.479 | 0.3840 | 3.200 |
| $31 / 2$ | 8 | 3.548 | 4.000 | 9.109 | 14.565 | 0.5136 | 4.280 |
| 4 | 8 | 4.026 | 4.500 | 10.790 | 11.312 | 0.6613 | 5.510 |
| 5 | 8 | 5.047 | 5.563 | 14.617 | 7.198 | 1.0393 | 8.660 |
| 6 | 8 | 6.065 | 6.625 | 18.974 | 4.984 | 1.5008 | 12.510 |
| 8 | 8 | 7.981 | 8.625 | 28.551 | 2.878 | 2.5988 | 21.680 |
| 10 | 8 | 10.020 | 10.750 | 40.483 | 1.826 | 4.0963 | 34.100 |
| 12 | 8 | 12.000 | 12.750 | 49.560 | 1.274 | 5.9036 | 49.000 |
| 14 | 8 | 13.250 | 14.000 | 54.570 | 1.046 | 7.1928 | 59.700 |
| 16 | 8 | 15.250 | 16.000 | 62.580 | 0.789 | 9.5301 | 79.100 |
| 18 | 8 | 17.250 | 18.000 | 70.590 | 0.617 | 12.1928 | 101.200 |

## Barlow's Formula

Pressure ratings of fluid vessels depend mainly on the tensile strength of the material being used at the process temperature, and the wall thickness of the vessel. Normally, the safety factor ratio should be at least 4 to 1 in determining the maximum pressure a vessel may see.
$\underset{\text { thickness (in) }}{\text { Minimum wall }}=\frac{\text { Maximum Pressure (PSI) } \times \text { OD of vessel (in) }}{2 \times \text { Tensile Strength (PSI) at process temperature }}$

